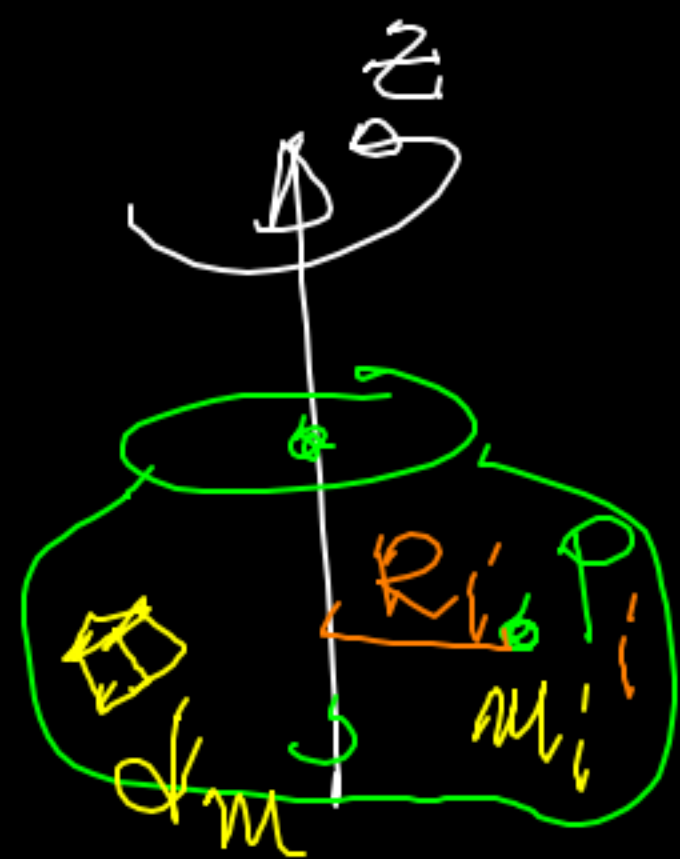


MOMENTO DI INERZIA

$$K = \frac{1}{2} I_z \omega_z^2$$



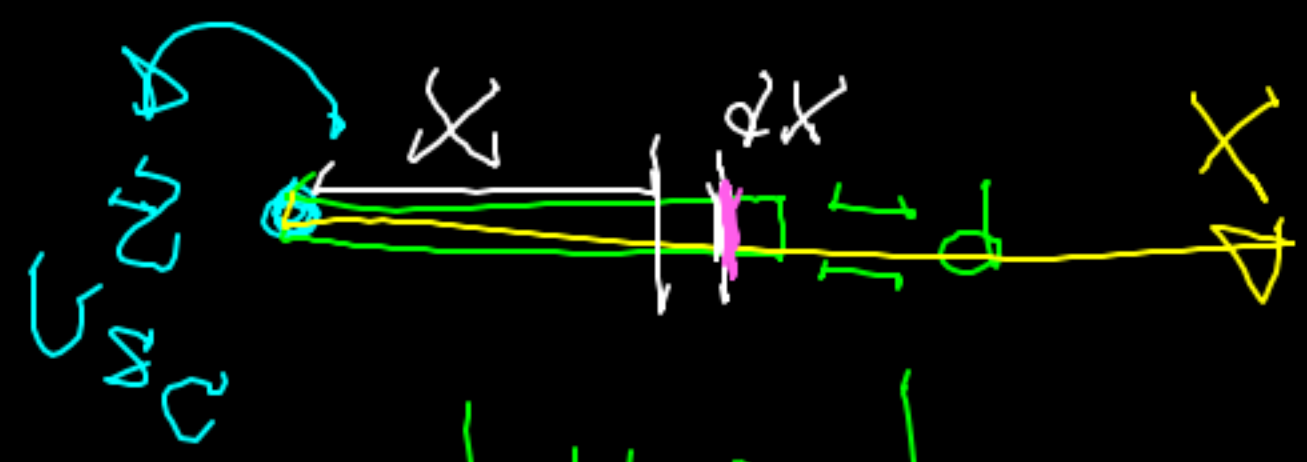
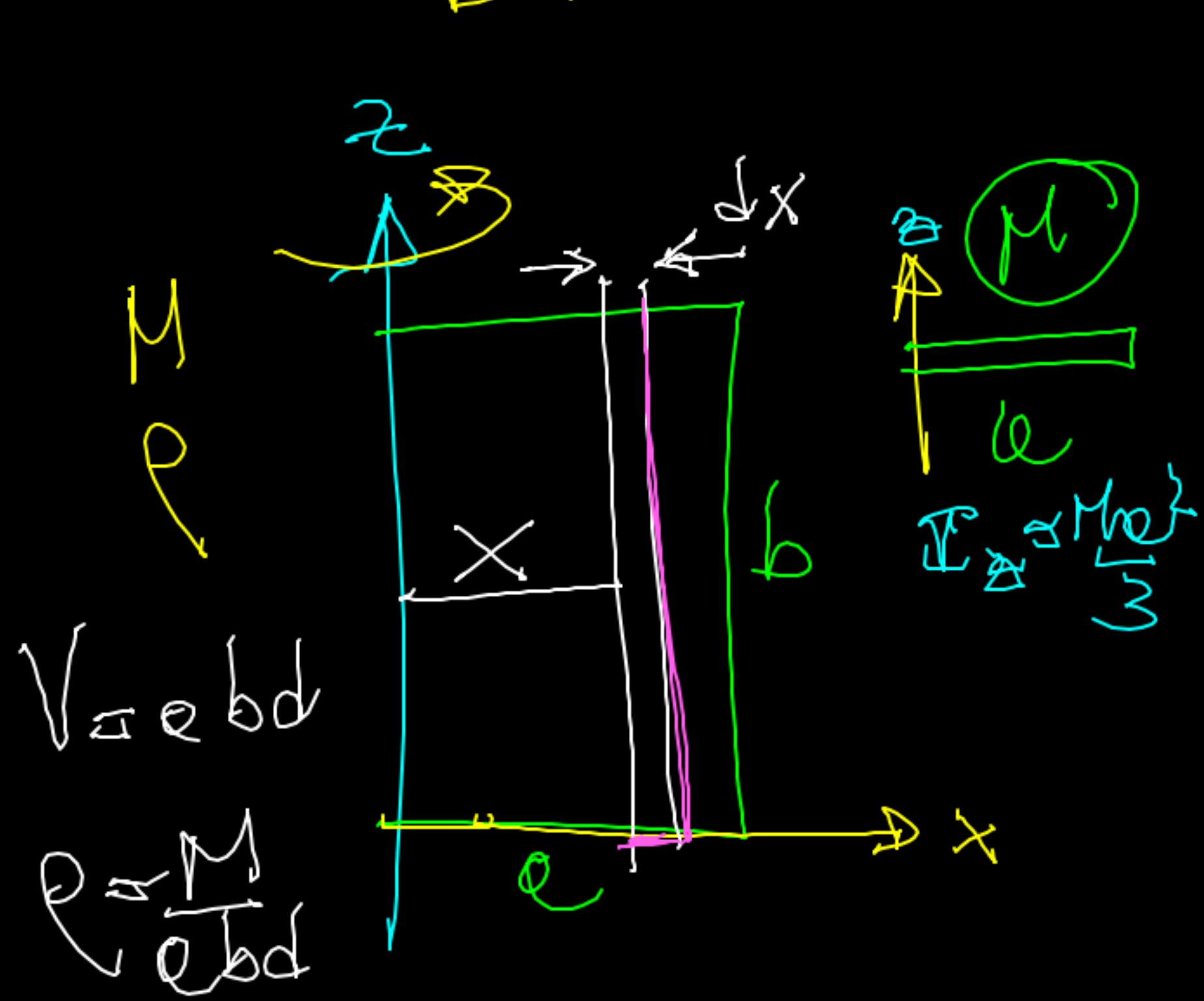
ROT. DI CORPO RIGIDO
INTORNO A Z FISSO

$$I_z = \sum_{i=1}^N m_i R_i^2$$

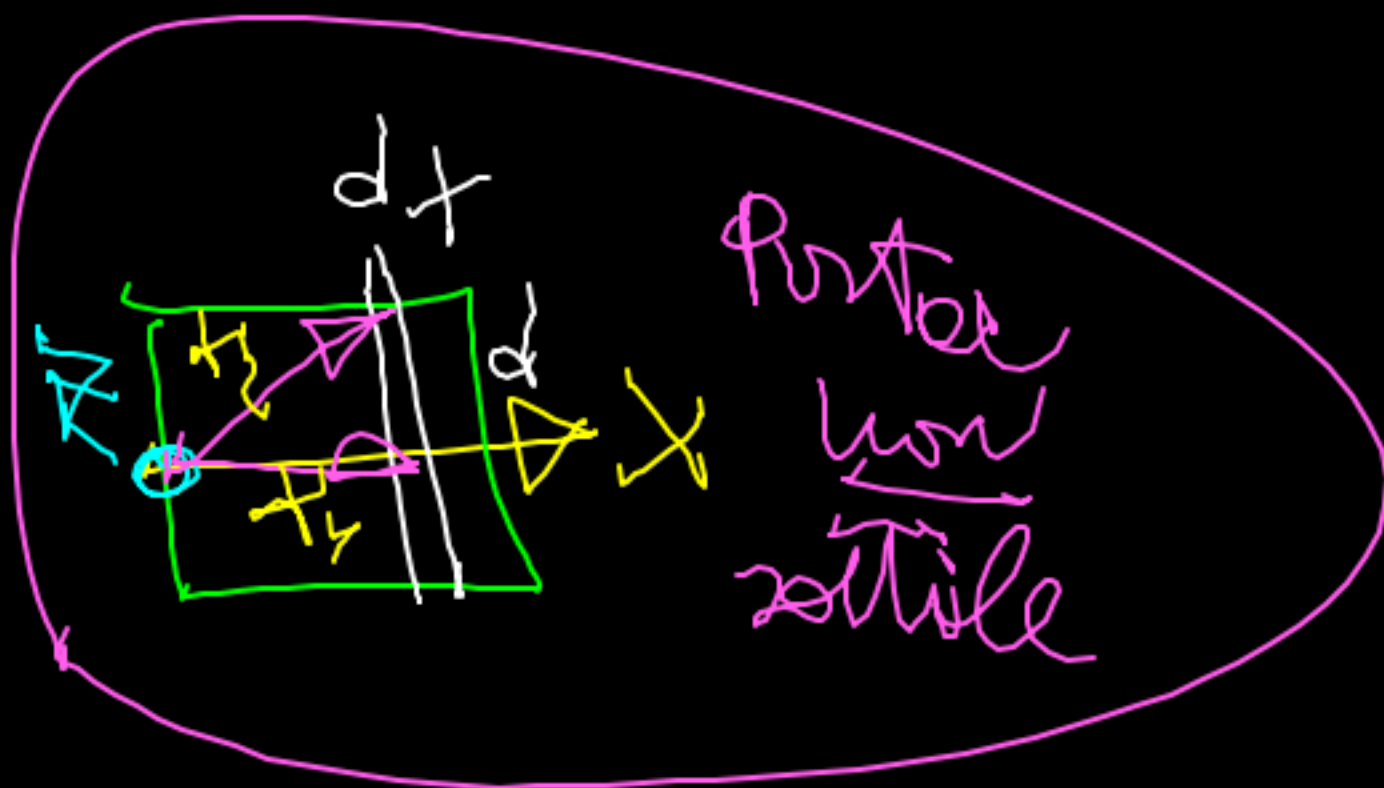
CASO
DISCRETO

$$I_z = \int_{\text{CORPO}} R^2 dm = \int_{\text{Vol. del corpo}} \rho R^2 dV$$

ESEMPIO DI CALCOLO



$d \ll a, b$
 parte rettangolare



I_z della parte (omogenea) rettangolare

$$I_z = \int \rho R^2 dV$$

Vol del corpo

$$dV = b d dx$$

$$R = x$$

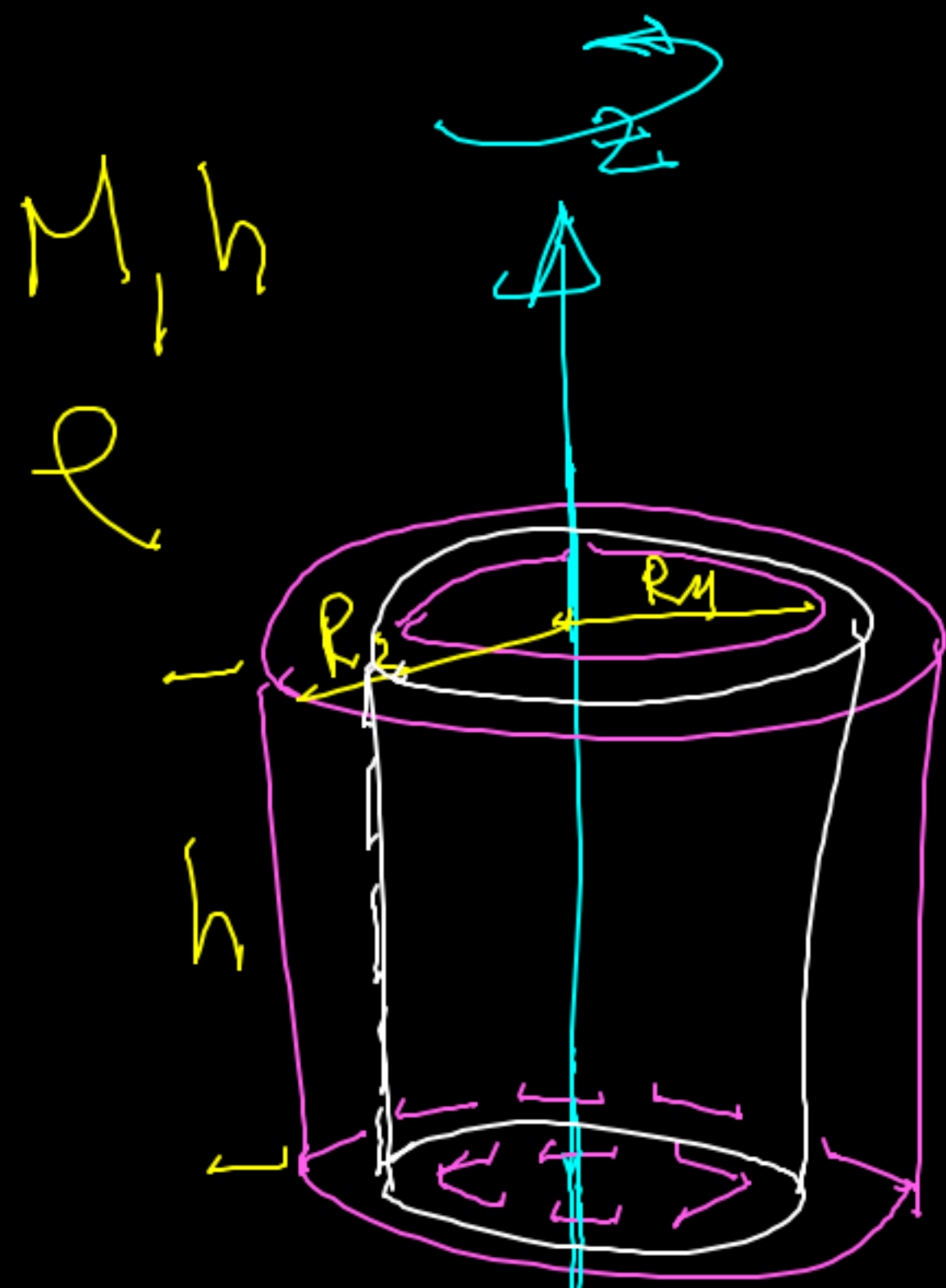
$$I_z = \int_0^e \rho b d x^2 dx$$

$$I_z = \rho b d \int_0^e x^2 dx =$$

$$= \rho b d \frac{e^3}{3} =$$

$$I_z = \frac{M}{e b d} b d \frac{e^3}{3} = \frac{M e^2}{3}$$

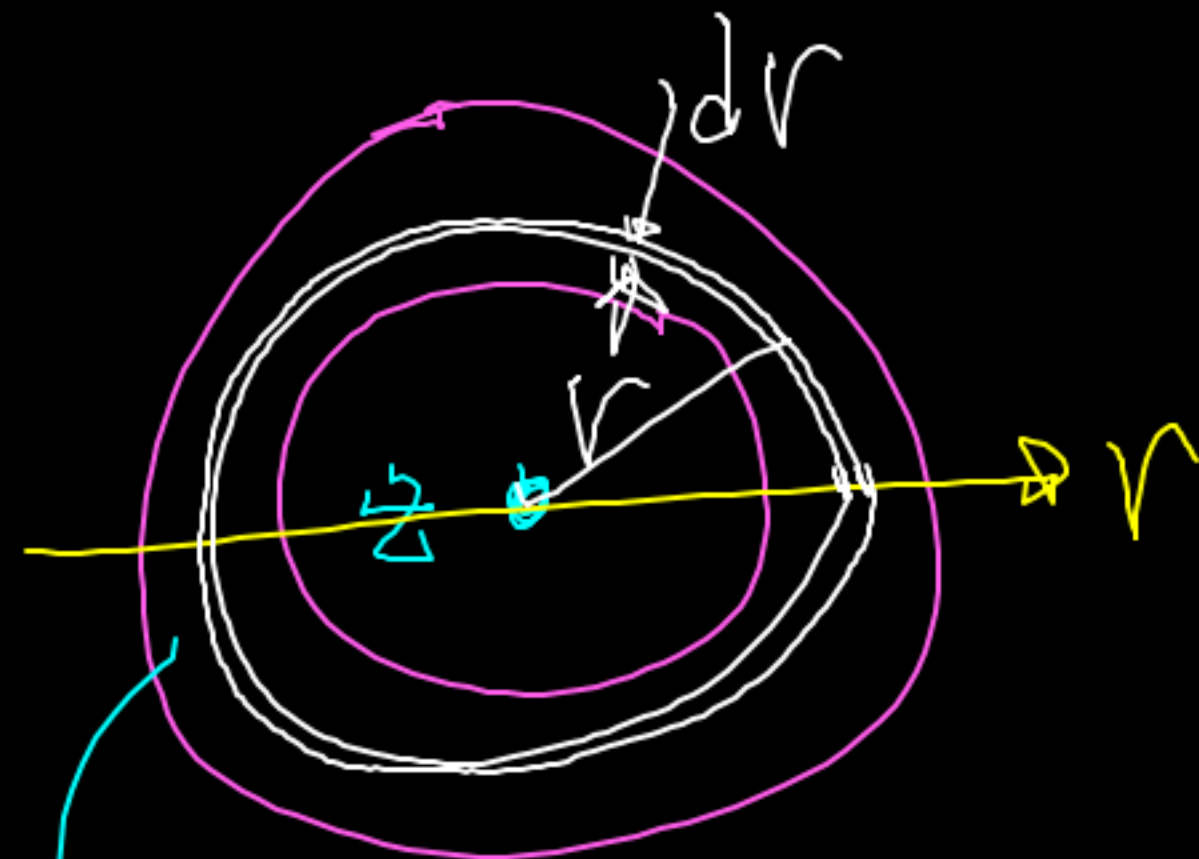
ESEMPIO MOM. DI INERZIA DEL CILINDRO (omogeneo)



$$I_z = \int_{\text{Vol}} \rho R^2 dV$$

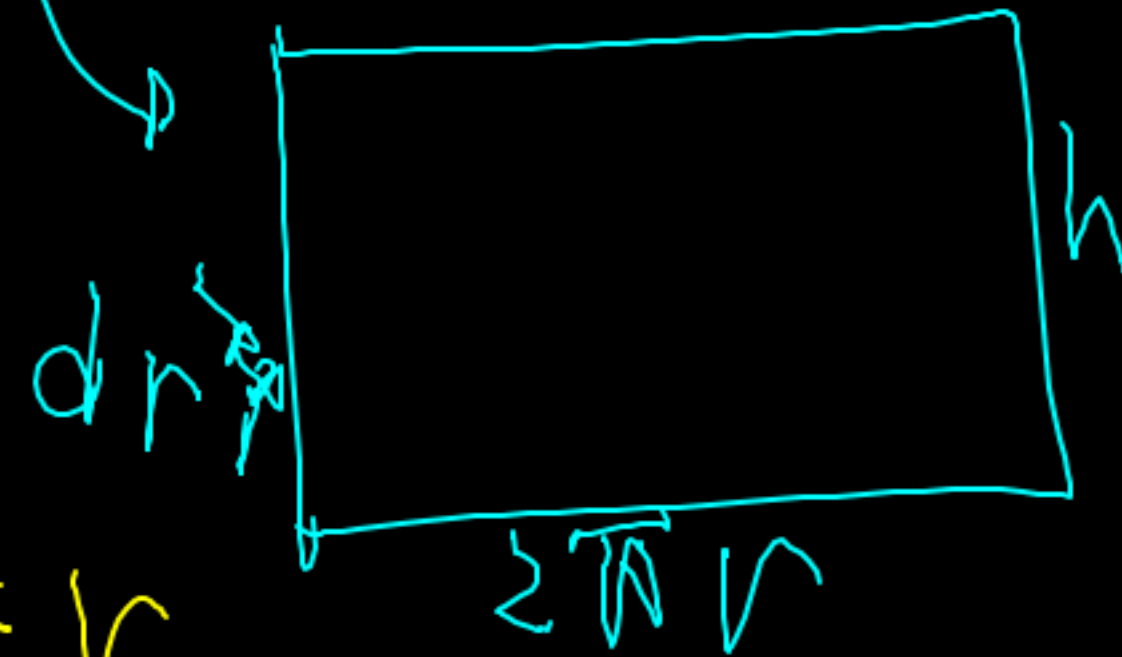
$$dV = 2\pi h r dr$$

$$R^2 = r^2$$

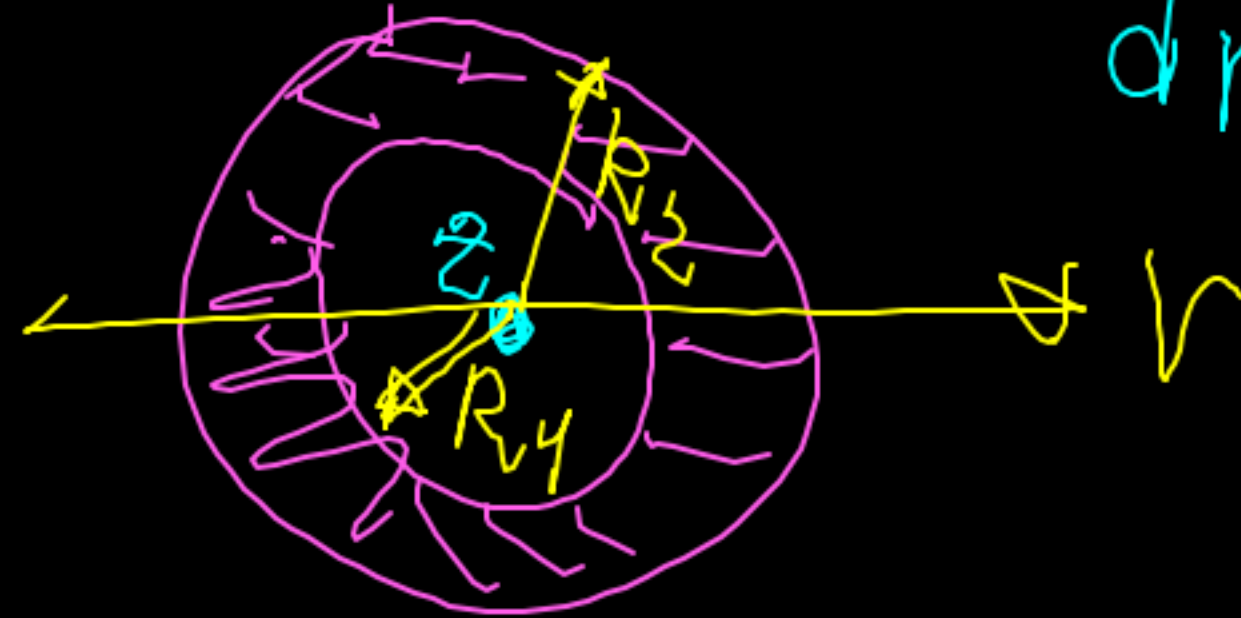


$$I_z = \int_{\text{Vol}} \rho 2\pi h r^3 dr =$$

$$= 2\pi h \rho \int_{R_1}^{R_2} r^3 dr = \frac{2\pi h \rho}{4} (R_2^4 - R_1^4)$$



$$\rho = \frac{M}{\text{Vol}} = \frac{M}{\pi R_2^2 h - \pi R_1^2 h}$$

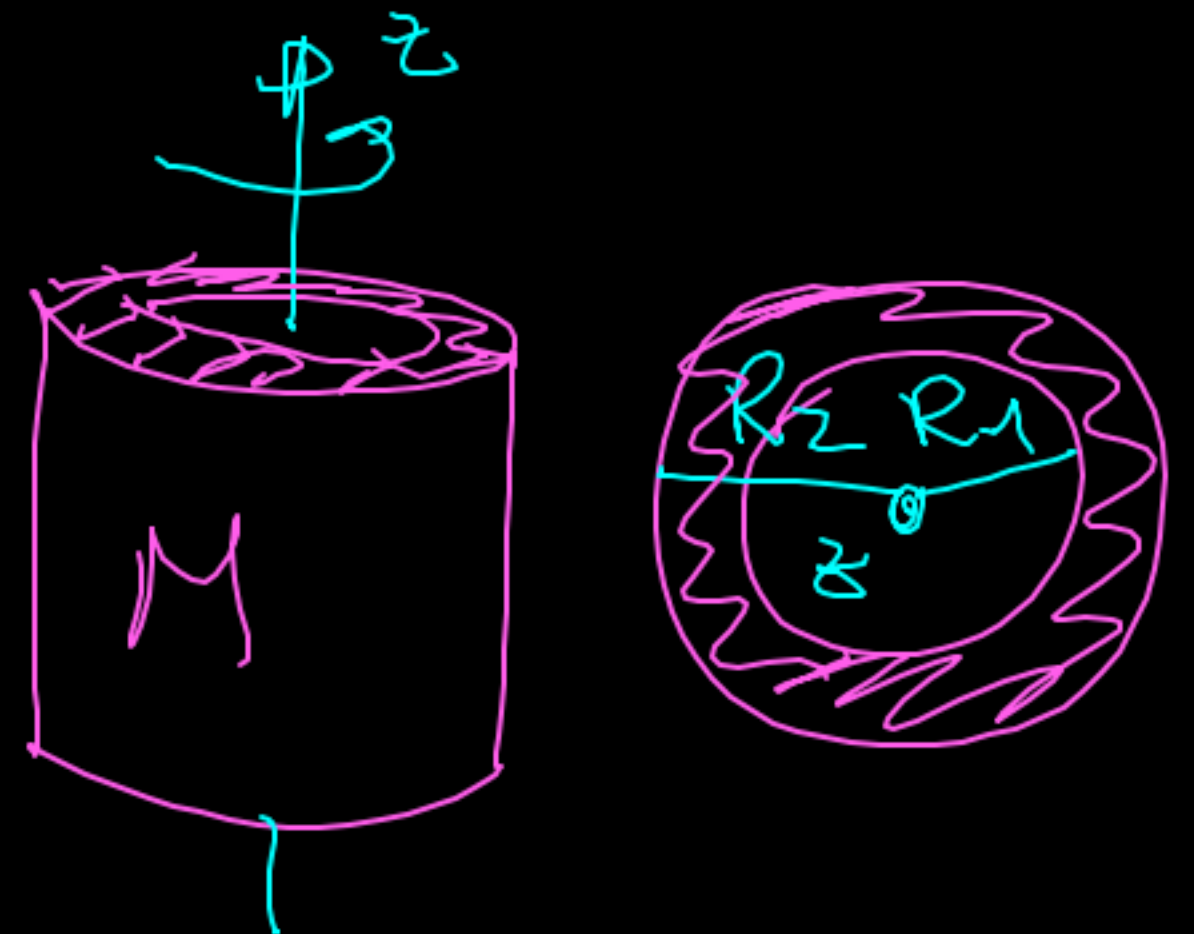


$$I_z = \frac{\pi h \rho}{2} (R_2^4 - R_1^4) = \frac{\pi h M (R_2^4 - R_1^4)}{2 (R_2^2 - R_1^2) \pi h} =$$

$$\frac{M}{2} \frac{(R_2^2 - R_1^2)(R_2^2 + R_1^2)}{(R_2^2 - R_1^2)} =$$

CIL. CAVO

$$I_z = \frac{M}{2} (R_2^2 + R_1^2)$$



NB

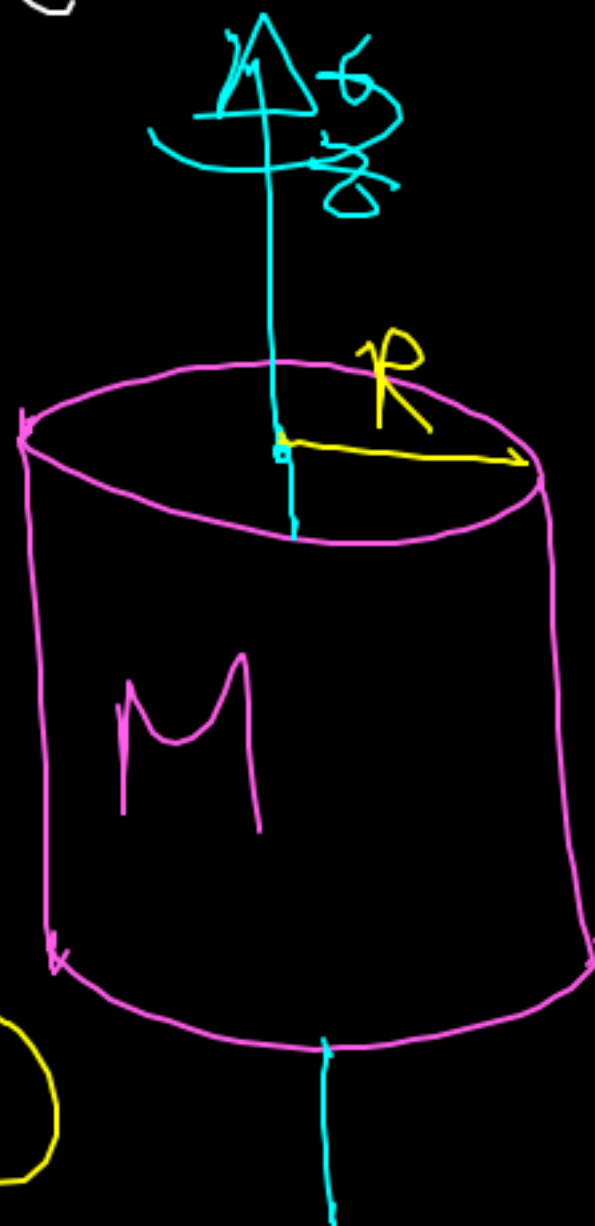
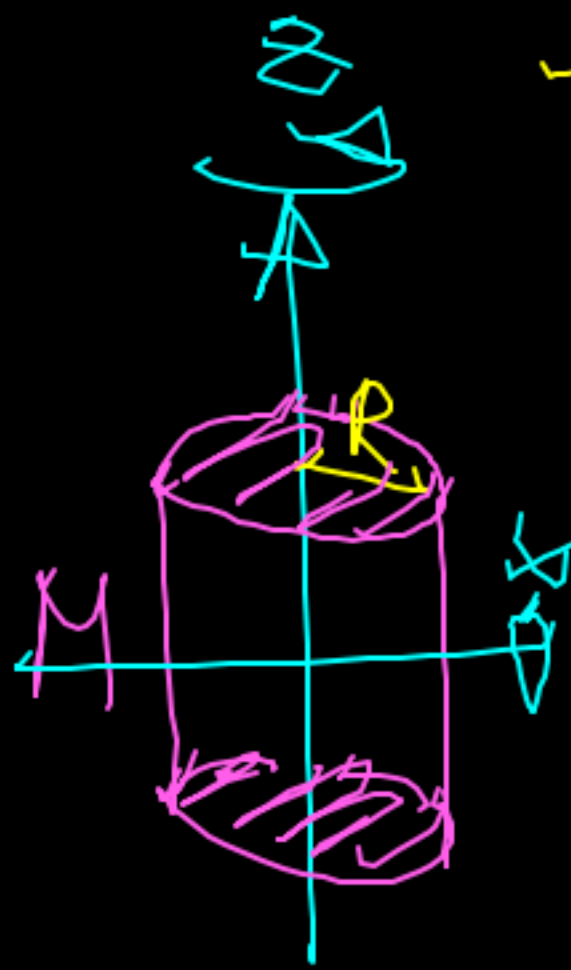
$$I_z \neq I_x$$

$$R_1 \rightarrow 0$$

$$R_2 \rightarrow R$$

$$I_z \approx \frac{1}{2} M R^2$$

CIL. PIENO (DISCO)



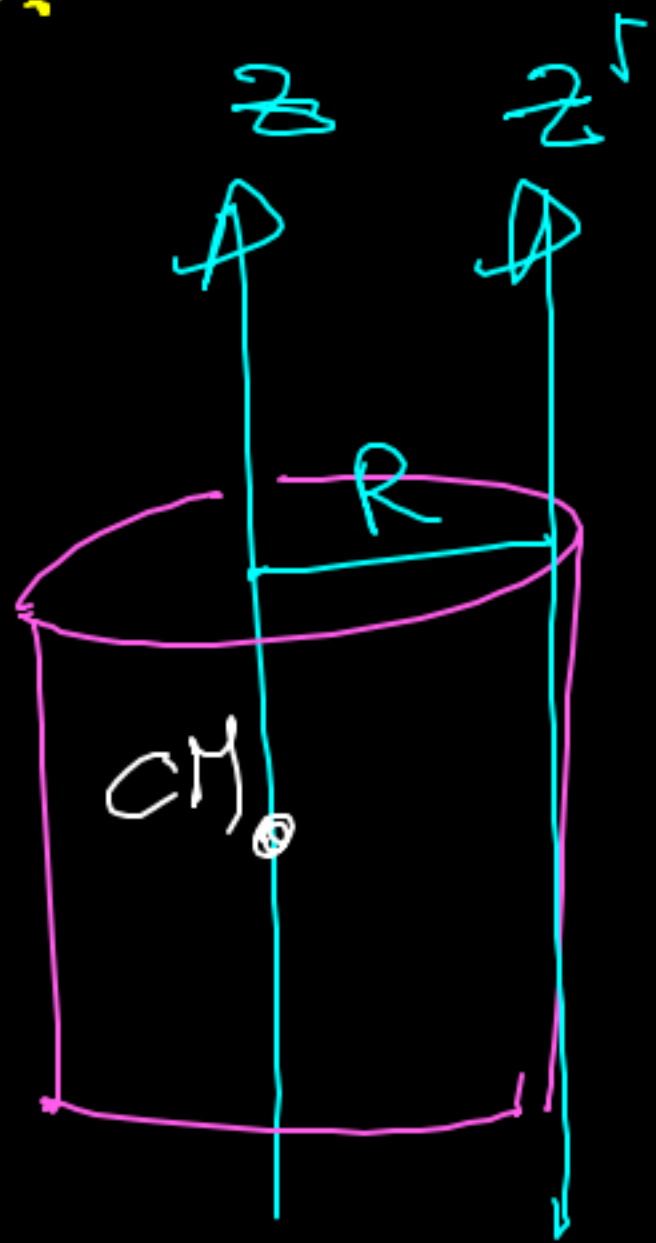
$$R_1 \rightarrow R$$

$$R_2 \rightarrow R$$

$$I_z = M R^2$$

STRATO CILIND. SOTTILE

ES.

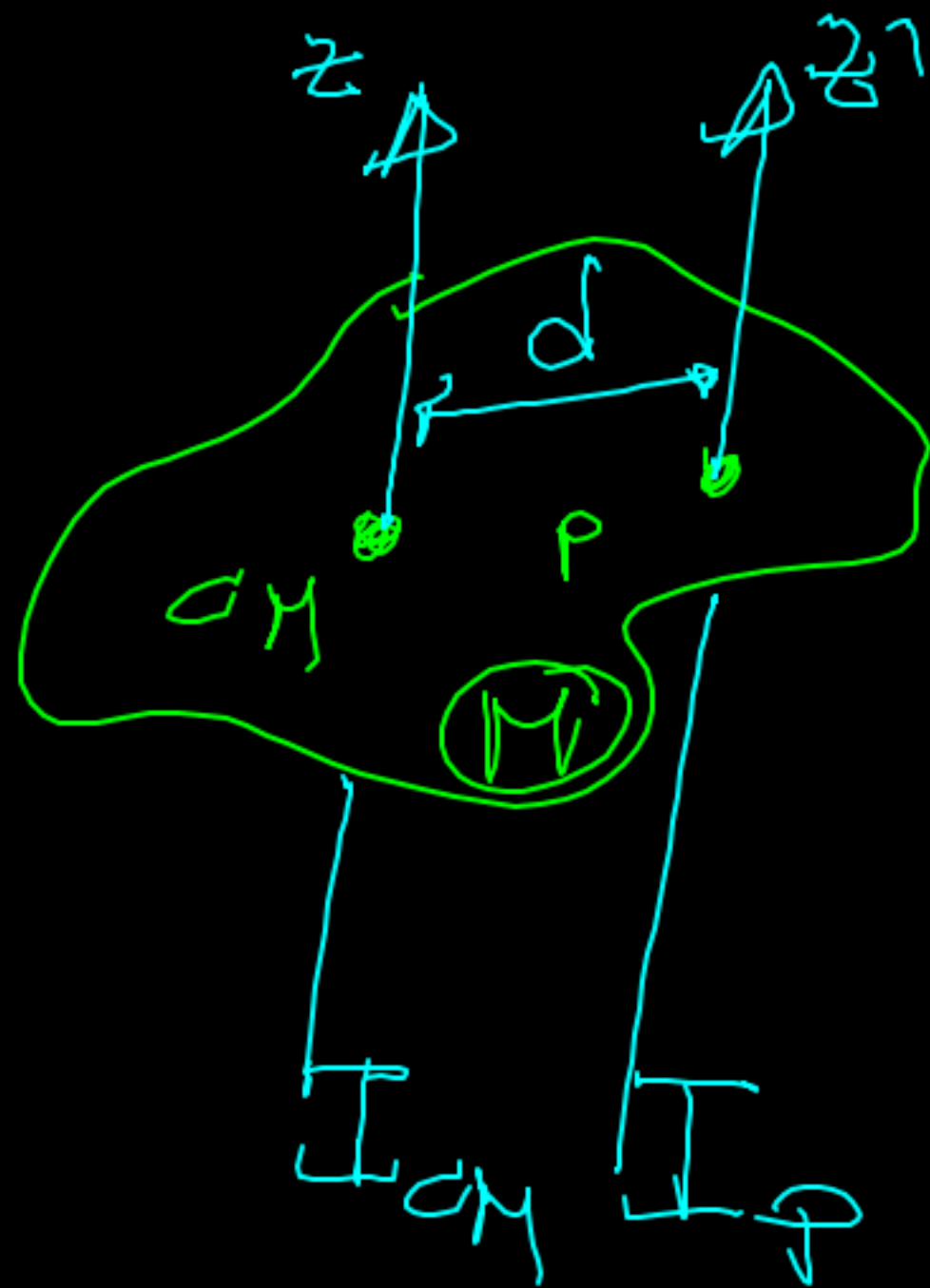


$$I_z \neq I_{z'}$$

$$I_z = \frac{MR^2}{2}$$

$$I_{z'} = \frac{MR^2}{2} + MR^2 = \frac{3}{2}MR^2$$

TEOREMA DI HUYGENS-STEINER O DEGLI ASSI PARALLELI



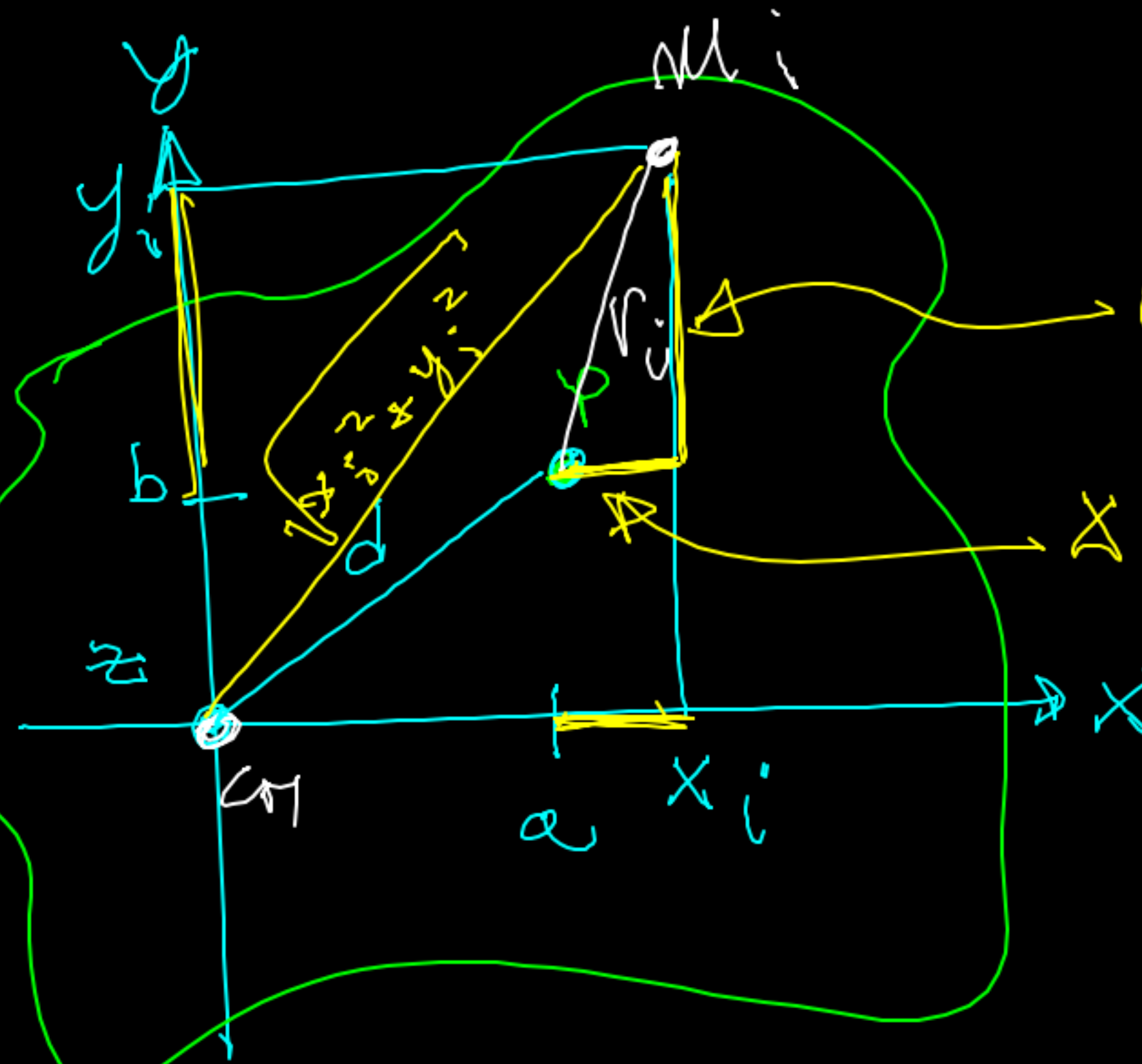
$z' \parallel z$, distanza d

$$I_P = I_{CM} + Md^2$$

ACHSE
Z-Mitte

D1 ROT

$I_{cm} + Md^2 = I_p$



$$I_p = \sum_{i=1}^N m_i r_i^2$$

$$r_i^2 = (x_i - a)^2 + (y_i - b)^2$$

$$x_i - a$$

$$r_i^2 = x_i^2 - 2ax_i + a^2 + y_i^2 - 2by_i + b^2$$

$$= (x_i^2 + y_i^2) - 2ax_i - 2by_i + a^2 + b^2$$

$x_{cm} = 0$
 $y_{cm} = 0$

$$d = \sqrt{a^2 + b^2}$$

$$x_{cm} = \frac{\sum m_i x_i}{M}$$

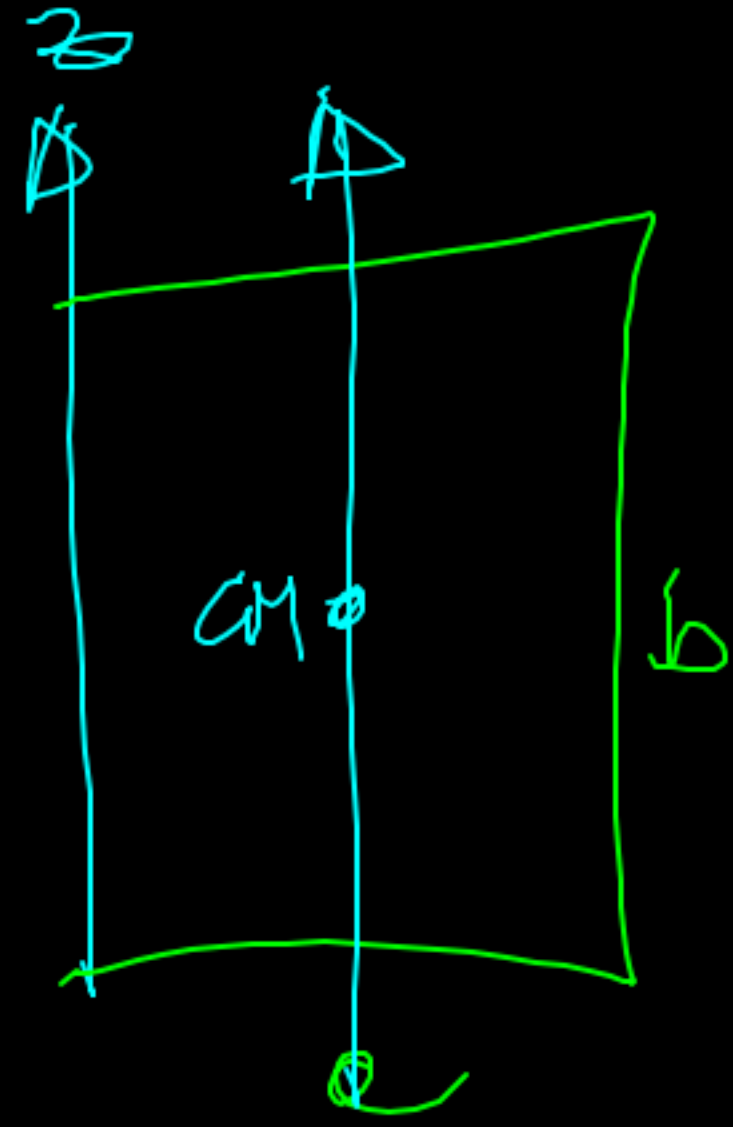
$$M x_{cm} = \sum m_i x_i = 0$$

$$I_p = \sum m_i (x_i^2 + y_i^2) + m_i (a^2 + b^2) - 2a m_i x_i - 2b m_i y_i$$

$$= \sum m_i \underbrace{(x_i^2 + y_i^2)}_{I_{cm}} + \sum m_i \underbrace{(a^2 + b^2)}_{d^2} - 2a \sum m_i x_i - 2b \sum m_i y_i$$

$$= I_{cm} + Md^2 - 2a \sum m_i x_i - 2b \sum m_i y_i$$

$\sum m_i x_i = 0$ $\sum m_i y_i = 0$



$$I_z = \frac{M a^2}{3}$$

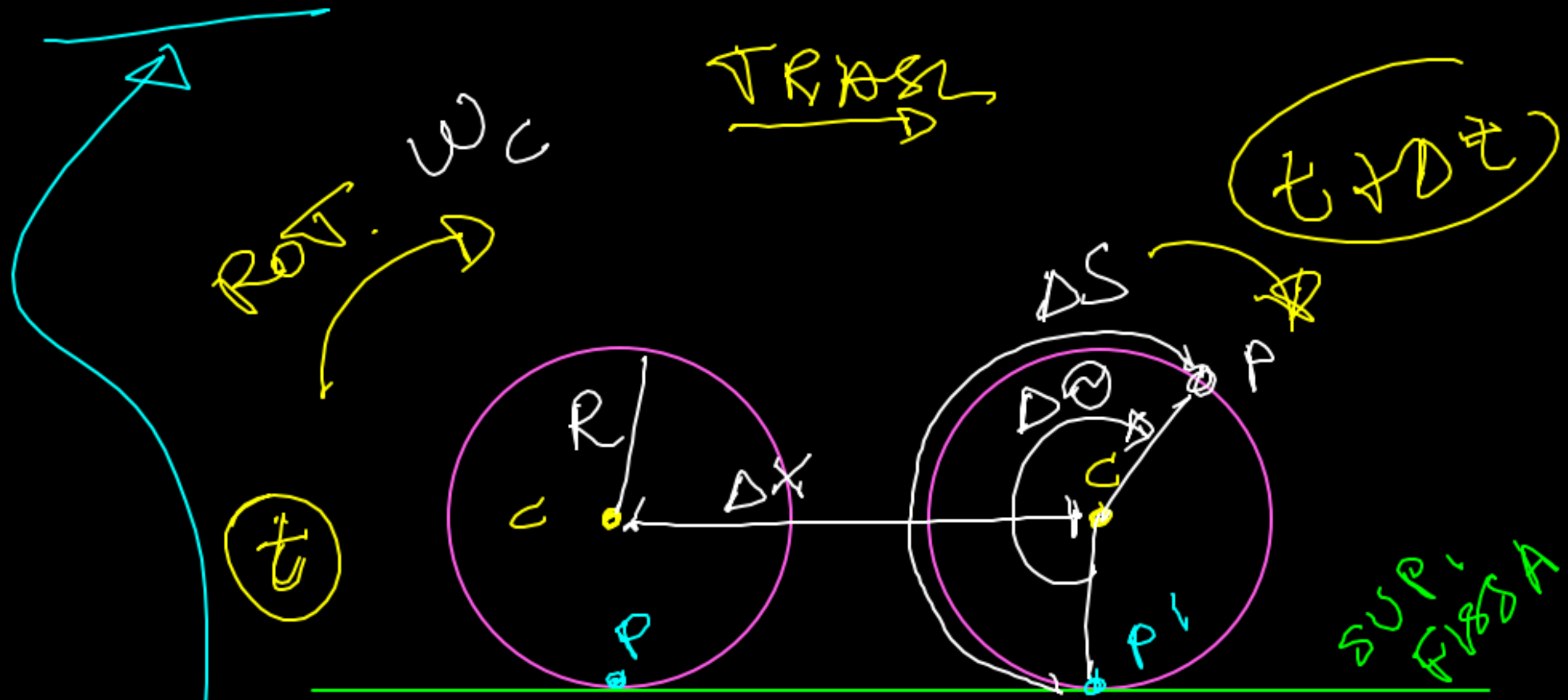
$$I_{cm} = ?$$

$$I_z = I_{cm} + M \frac{a^2}{4}$$

$$I_{cm} = I_z - M \frac{a^2}{4} =$$

$$= \frac{M a^2}{3} - M \frac{a^2}{4} = M a^2 \frac{1}{12}$$

PURO ROTOLAMENTO DI UN CORPO RIGIDO



$$v_c = \omega_c R$$

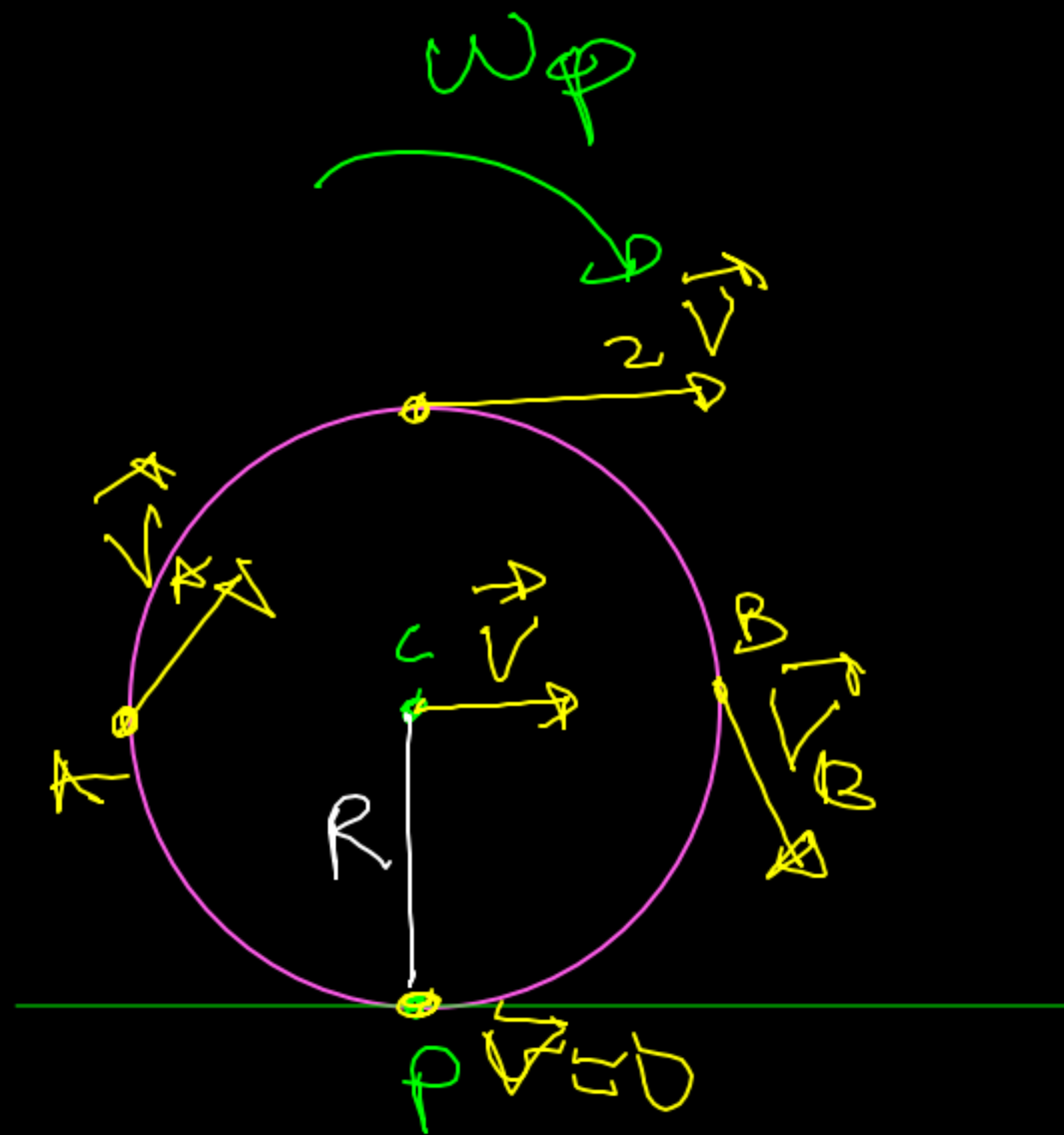
Condizione di puro rotolamento (l'axe pesante per C rimane parallelo e scivola lungo il moto)

$v_p(t) = 0$
 " ROTOLARE SENZA STRISCIARE "

$$\frac{\Delta S}{\Delta t} = \frac{\Delta X}{\Delta t} = v_c$$

$$\frac{\Delta S}{R} = \Delta \theta \Rightarrow \frac{R \Delta \theta}{\Delta t} = R \omega_c$$

VISTA
DA
 \hat{P}^n



ROTAZIONE
ISTANTANEA
INTORNO A P

$$V = \omega_P R \quad \text{"C" visto da "P"} \quad \Rightarrow \quad \omega_C = \omega_P = \omega$$

$$V = \omega_C R \quad \text{"P" visto da "C"}$$

Se puro
rotolamento

ENERGIA CINETICA (puramente rotazionale)

$$K = \frac{1}{2} I_p \omega^2 = \frac{1}{2} (I_{cm} + MR^2) \omega^2 =$$

CECM $\Rightarrow \frac{1}{2} I_{cm} \omega^2 + \frac{1}{2} MR^2 \omega^2 = \frac{1}{2} I_{cm} \omega^2 + \frac{1}{2} M v_{cm}^2$

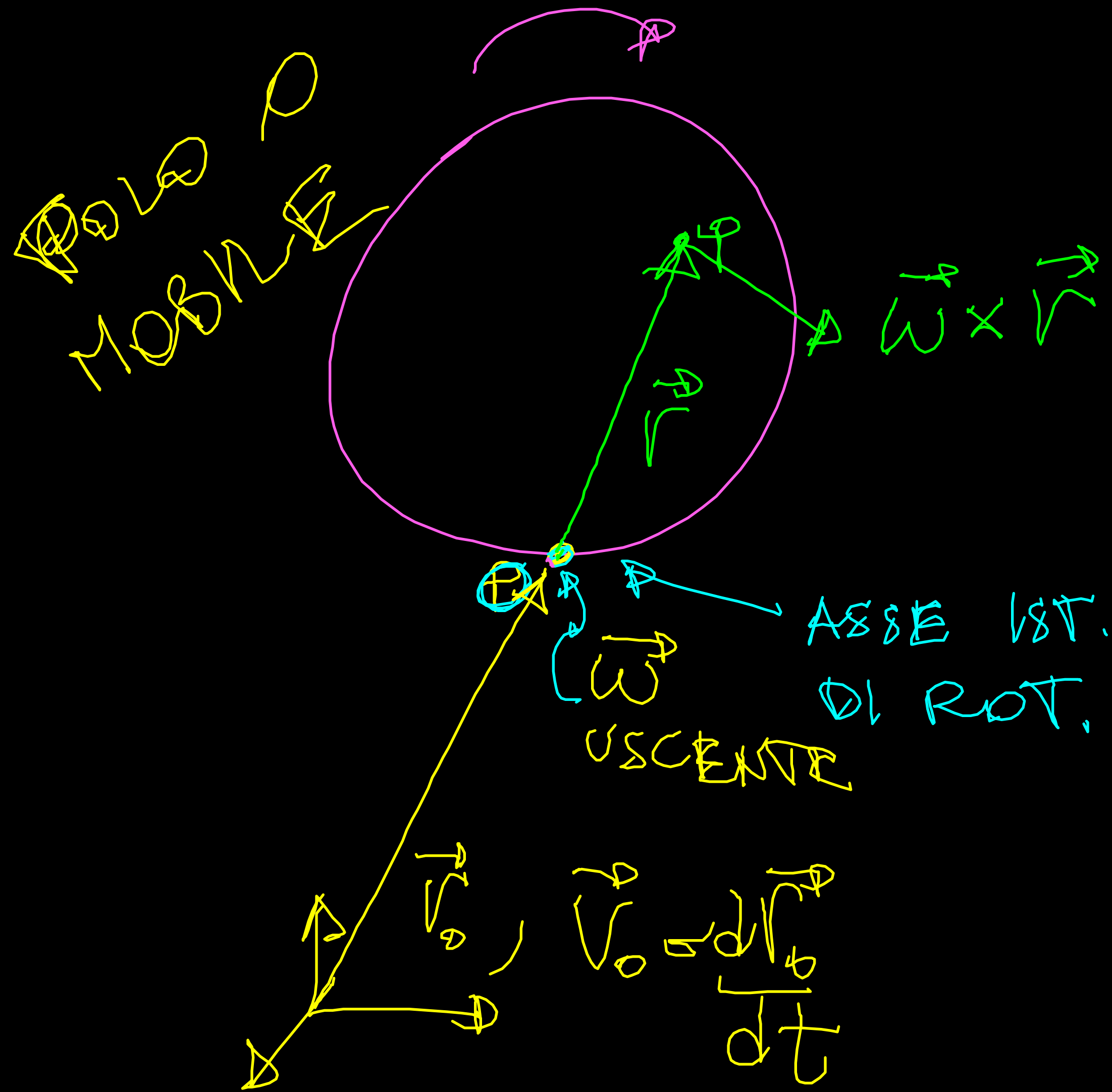
$$v_{cm} = \omega R$$

$$K = \underbrace{\frac{1}{2} I_{cm} \omega^2}_{E.c. \text{ rot.}} + \underbrace{\frac{1}{2} M v_{cm}^2}_{E.c. \text{ trad.}}$$

Se no puramente rot.

$$v \neq \omega R \Rightarrow K = \frac{1}{2} I_{cm} \omega^2 + \frac{1}{2} M v^2$$

TEOREMA DI KÖNIG
(ASSE DI ROT. SI MANTIENI)



$$\vec{V}_P = \vec{V}_0 + \vec{\omega} \times \vec{r}$$

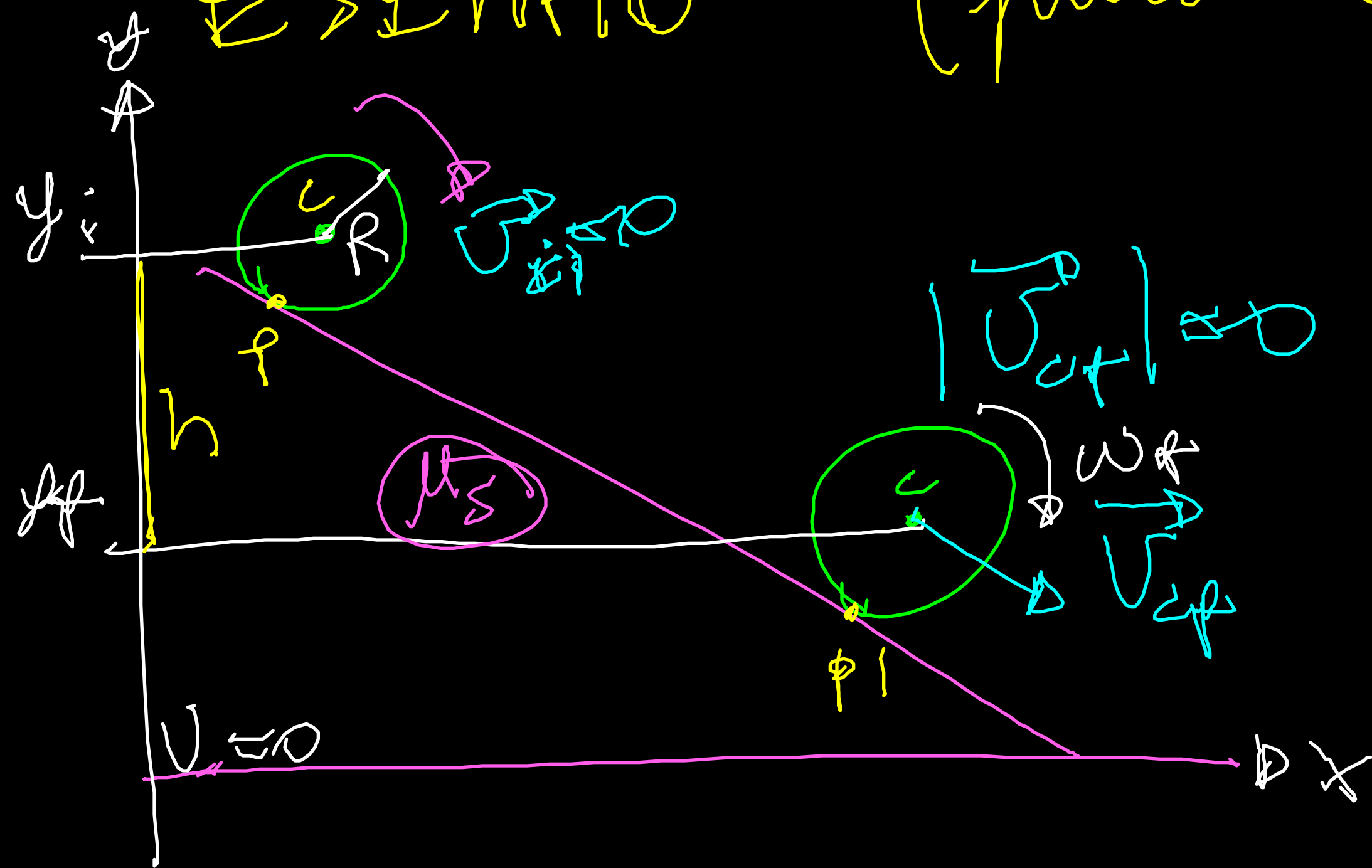
$$\vec{V}_0$$

Vel. di P
vista da
un sistema
fisso

Vel. del polo
mobile visto
dal sistema fisso

ESEMPLO

(puro rotolamento)



$$K_f + U_f = K_i + U_i$$

$$\frac{1}{2} M v_{CM}^2 + \frac{1}{2} I_{CM} \omega_f^2 + M g y_f = M g y_i$$

$$\omega_f = \frac{v_{CM}}{R}$$

$$\frac{1}{2} M v_{CM}^2 = M g (y_f - y_i) - \frac{1}{2} I_{CM} \omega_f^2$$

$$v_{CM}^2 + \frac{2 I_{CM} v_{CM}^2}{2 M R^2} = \frac{2 M g (y_f - y_i)}{M}$$

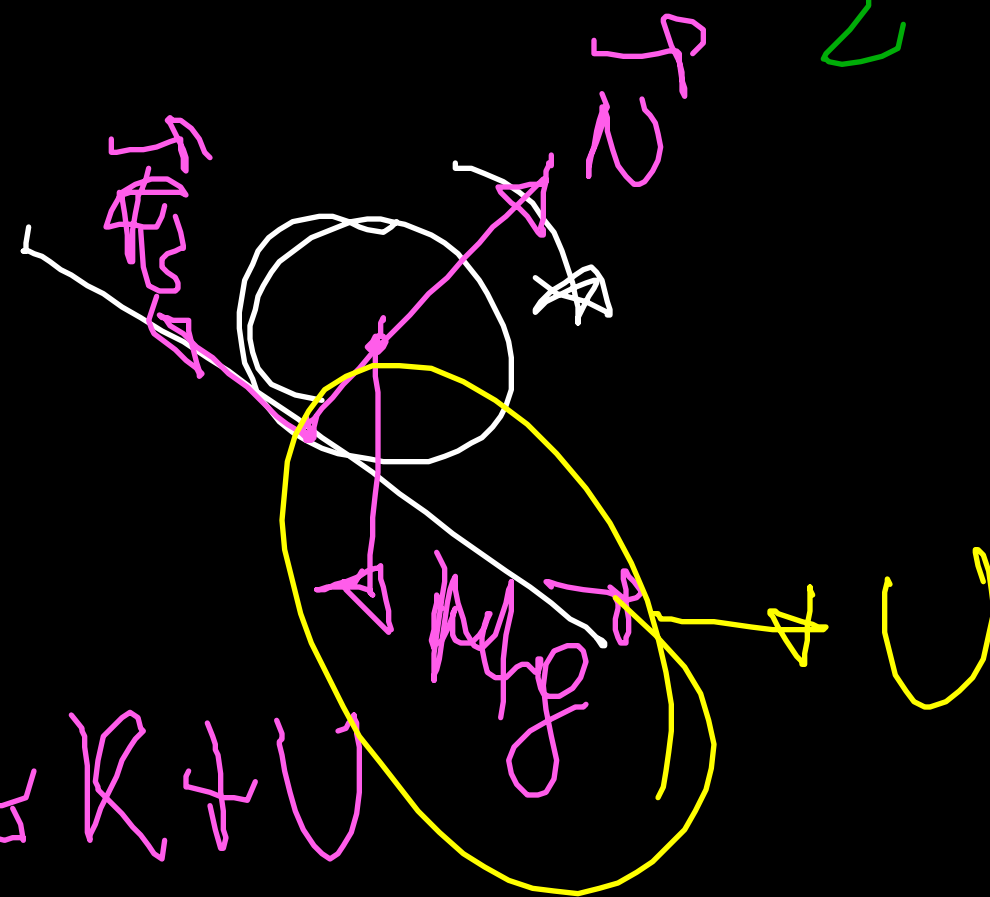
$$v_{CM}^2 \left(1 + \frac{I_{CM}}{M R^2} \right) = 2 g (y_f - y_i)$$

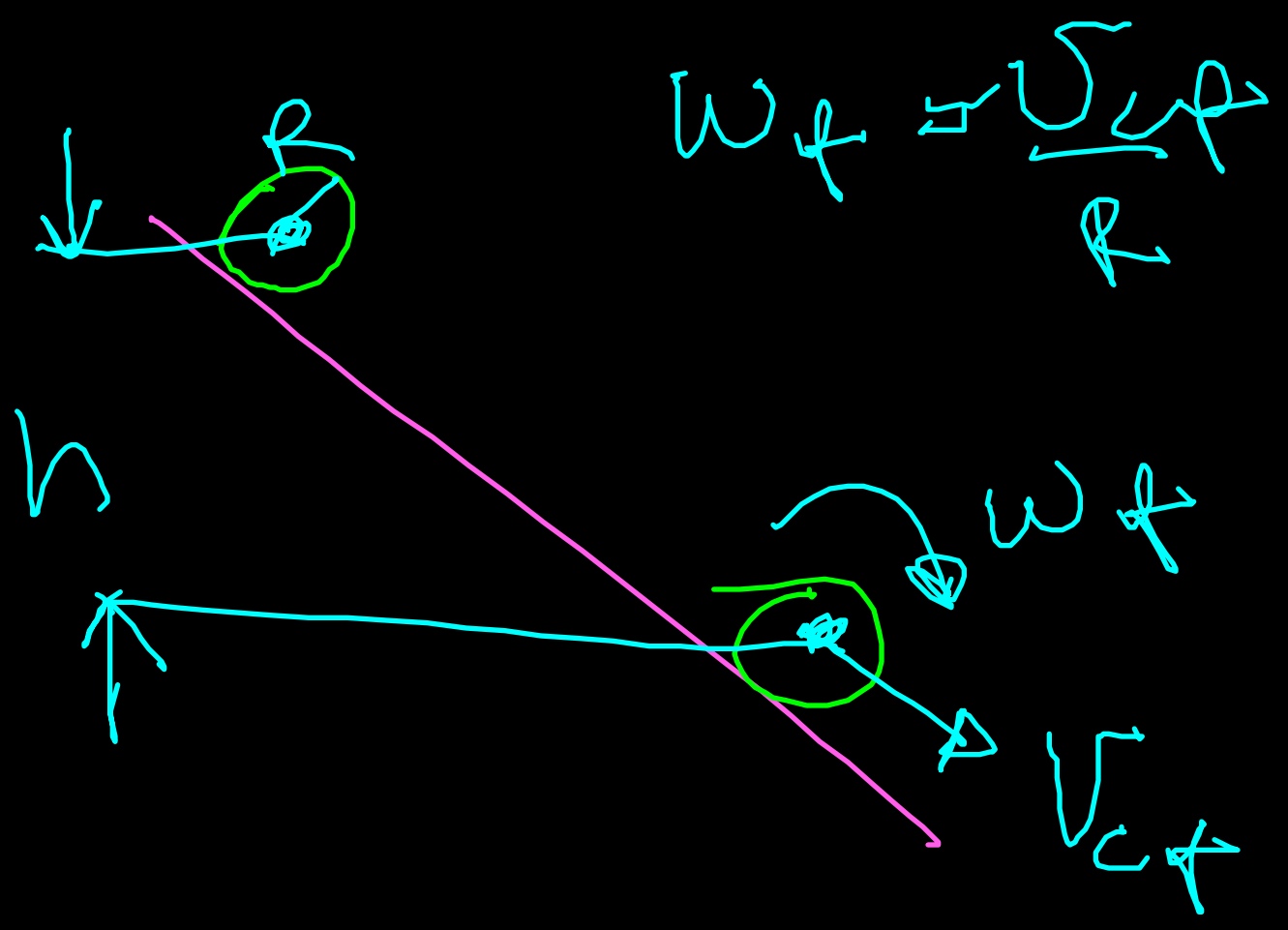
$$v_{CM} = \sqrt{2 g h / \left(1 + \frac{I_{CM}}{M R^2} \right)}$$

puro rotolamento senza
slittamenti CECM

$$L_{attr.} = 0 \Rightarrow E = K + U$$

e cost.





$$v_{cm} = \sqrt{\frac{2gh}{1 + \frac{I_{cm}}{MR^2}}}$$

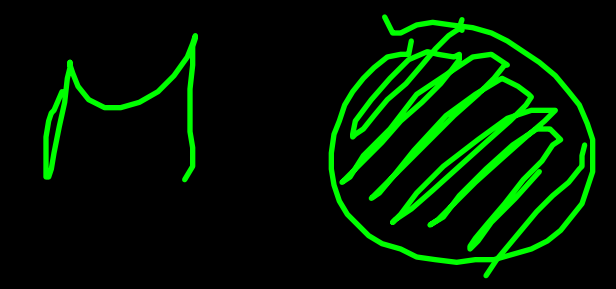
CILINDRO $\rightarrow I_{cm} = \frac{1}{2} MR^2$

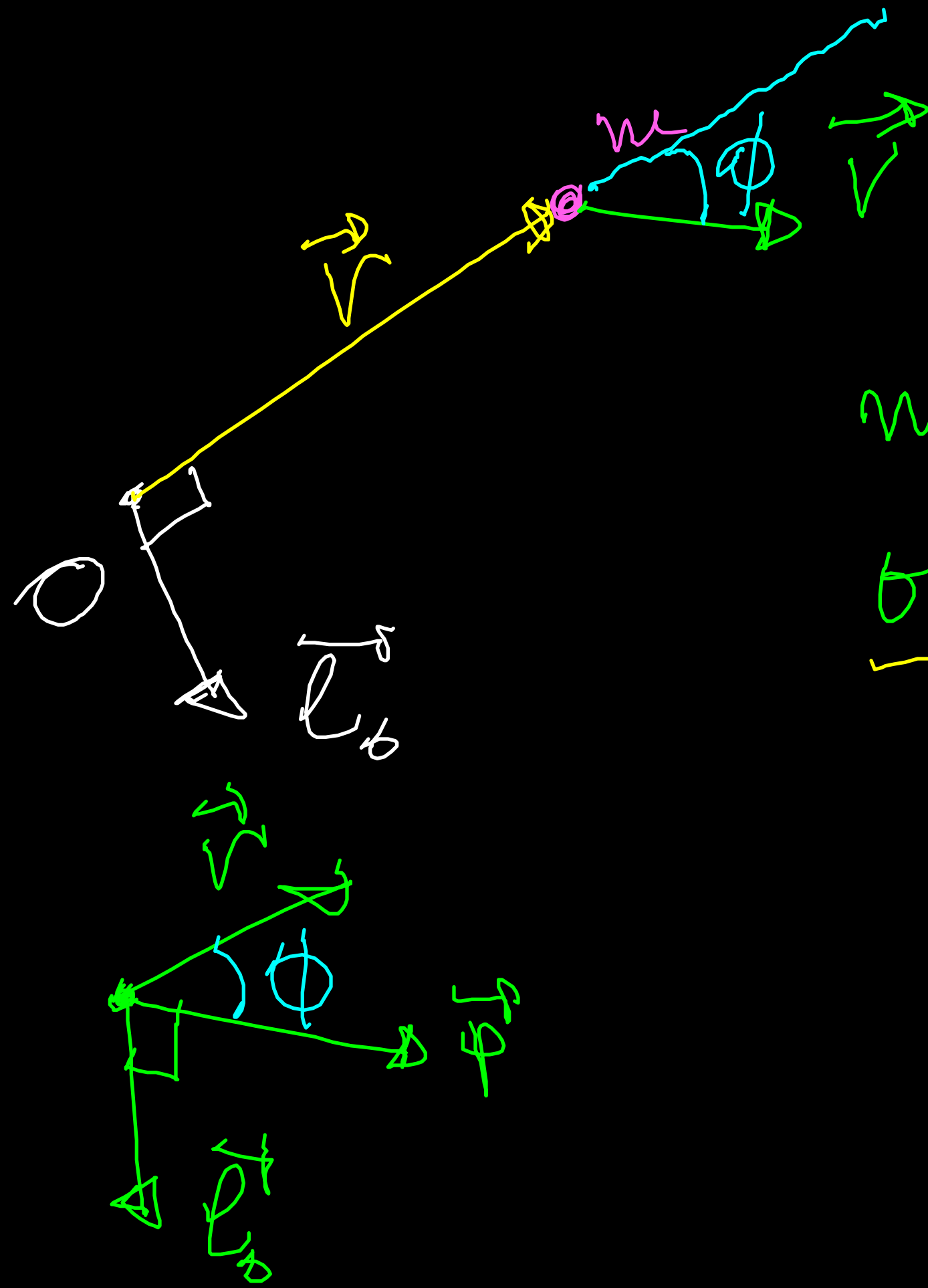
$$v_{cil} = \sqrt{\frac{2gh}{1 + \frac{1}{2}}} = \sqrt{\frac{4gh}{3}}$$

ANELLO $\rightarrow I_{cm} = MR^2$

$$v_{anello} = \sqrt{\frac{2gh}{1+1}} = \sqrt{gh}$$

$v_{cil} > v_{anello}$





$$\vec{p} = m\vec{v}$$

momento della quantità di moto
o momento angolare

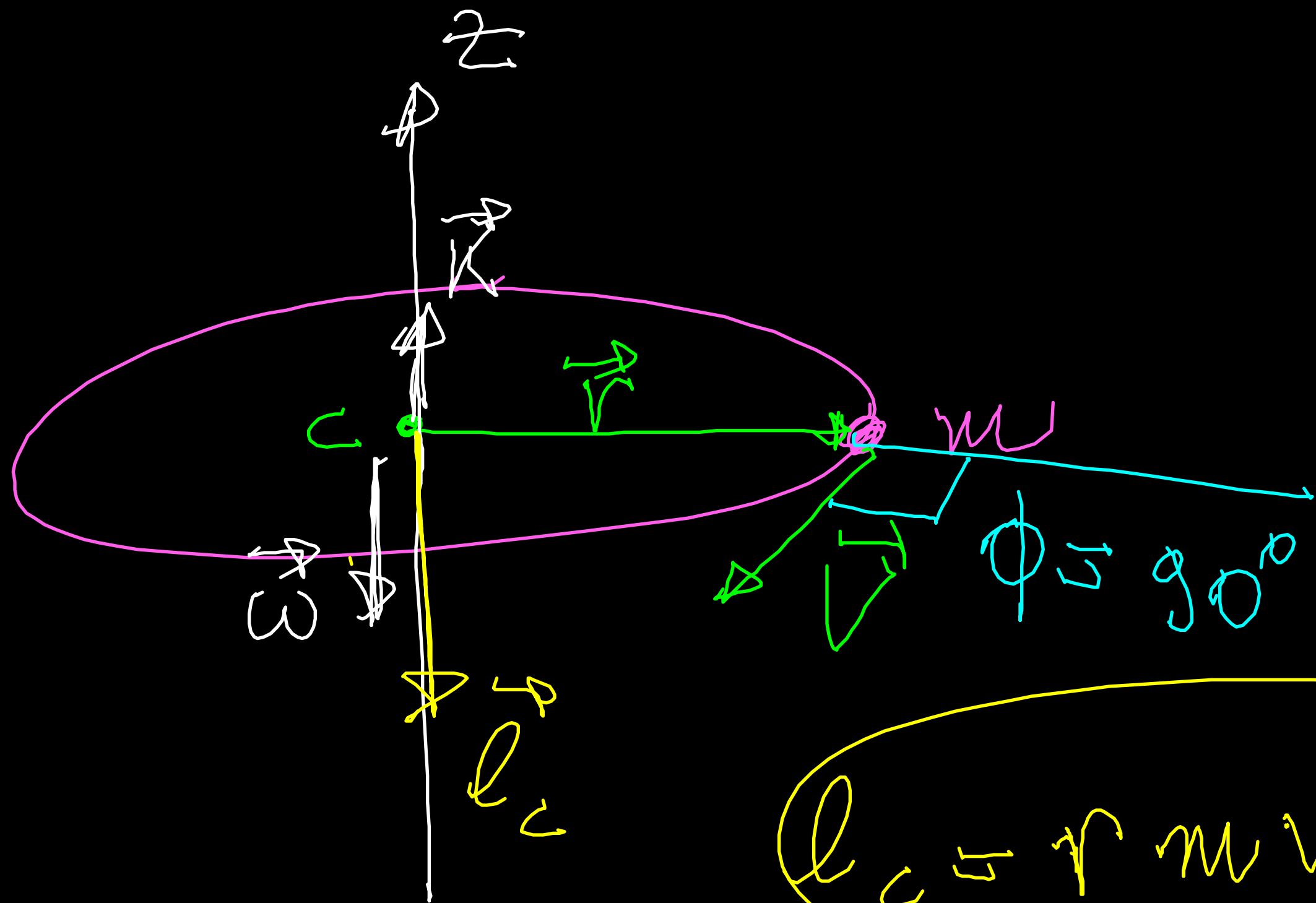
$$\vec{l}_O = \vec{r} \times \vec{p}$$

O ← polo

$$l_O = r p \sin\phi$$

MOTO CIRCOLARE

→ MOM. ANGOLA



$$v = \omega r$$

Se moto circ. unif

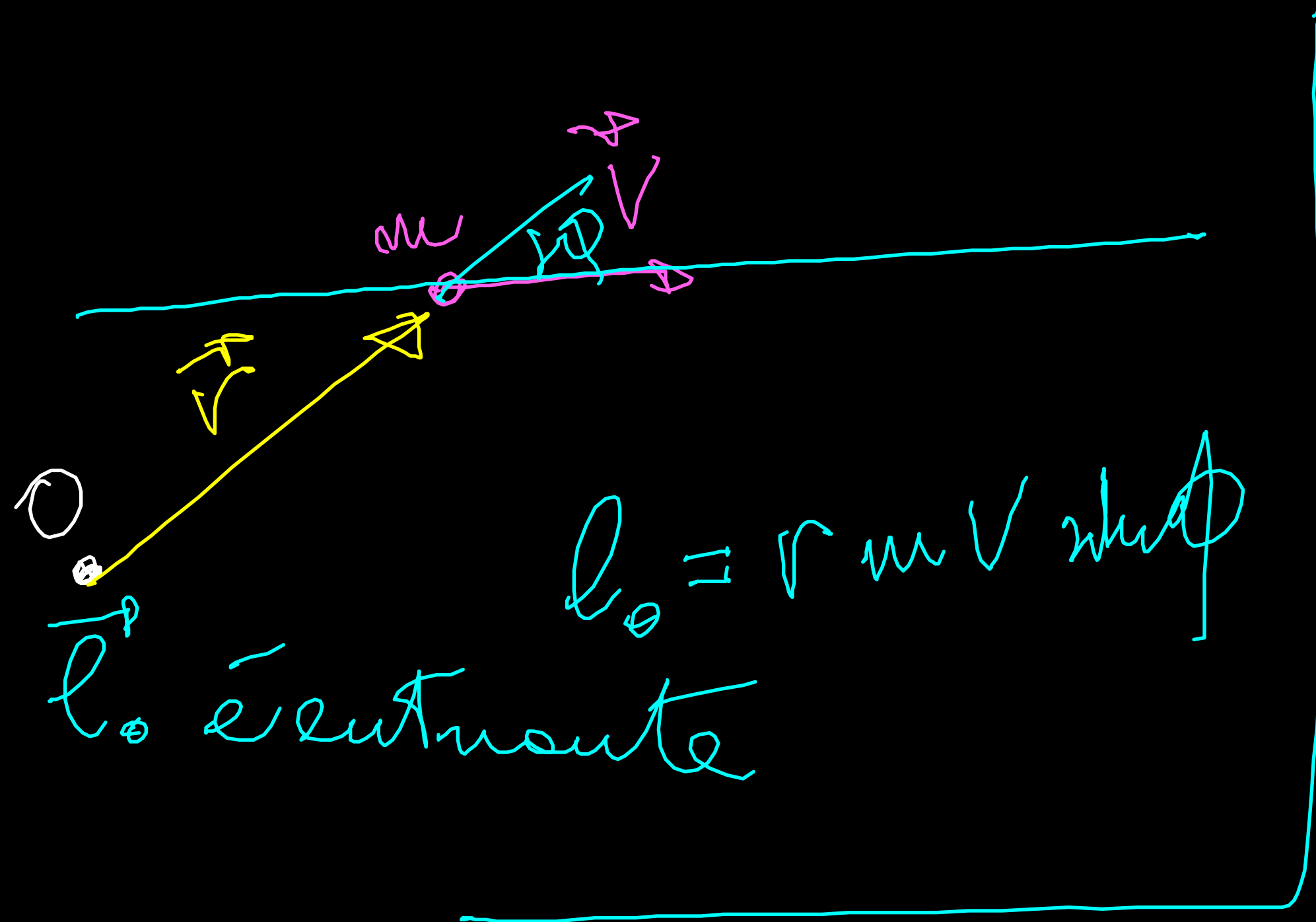
$\vec{\omega}$ é constante

\vec{l}_c é constante

$$l_c = r m v = m \omega r^2$$

$$\vec{l}_c = l_z \hat{k} = (-m \omega r^2) \hat{k}$$

$$\vec{\omega} = \omega_z \hat{k} = \left(-\frac{v}{r}\right) \hat{k}$$



$$\sum \vec{F} = \frac{d\vec{p}}{dt}$$

(punto materiale)

$$\frac{d\vec{l}_0}{dt} = \frac{d}{dt} (\vec{r} \times \vec{p})$$

Vettore
posizionale
 $O \equiv$ origine
 O è fisso

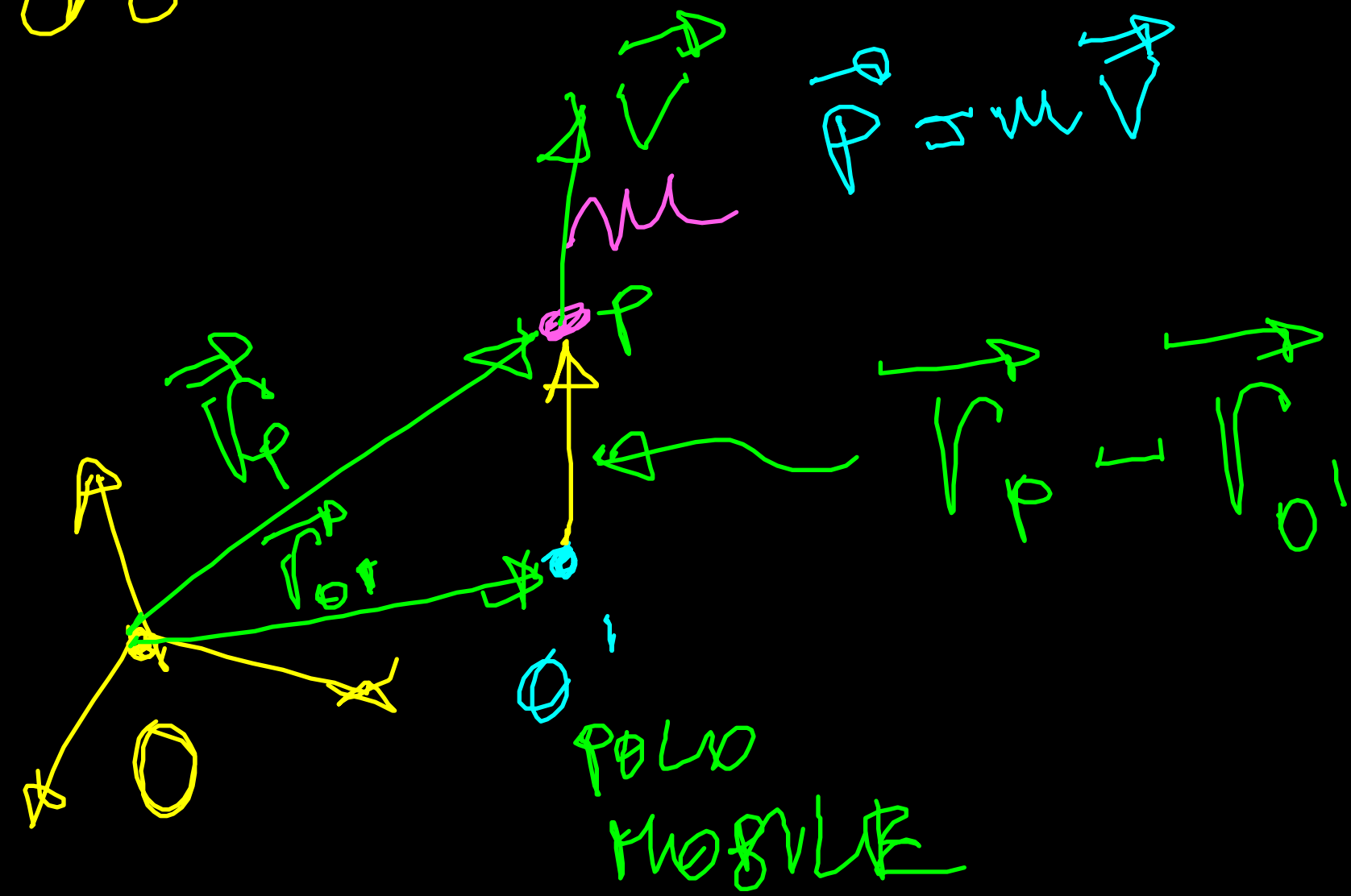
$$\begin{aligned} \frac{d\vec{l}_0}{dt} &= \frac{d\vec{r}}{dt} \times \vec{p} + \vec{r} \times \frac{d\vec{p}}{dt} = \\ &= \cancel{\vec{v} \times (m\vec{v})} + \vec{r} \times \left(\sum \vec{F} \right) \end{aligned}$$

$$\frac{d\vec{l}_0}{dt} = \vec{r} \times \left(\sum \vec{F} \right)$$

momento delle forze applicate

$$\frac{d\vec{l}_O}{dt} = \vec{r}_P \times \left(\sum \vec{F}_i \right)$$

- polo O fissa e dell'origine
- S.d.R. inerziale



S.d.R. INERZIA

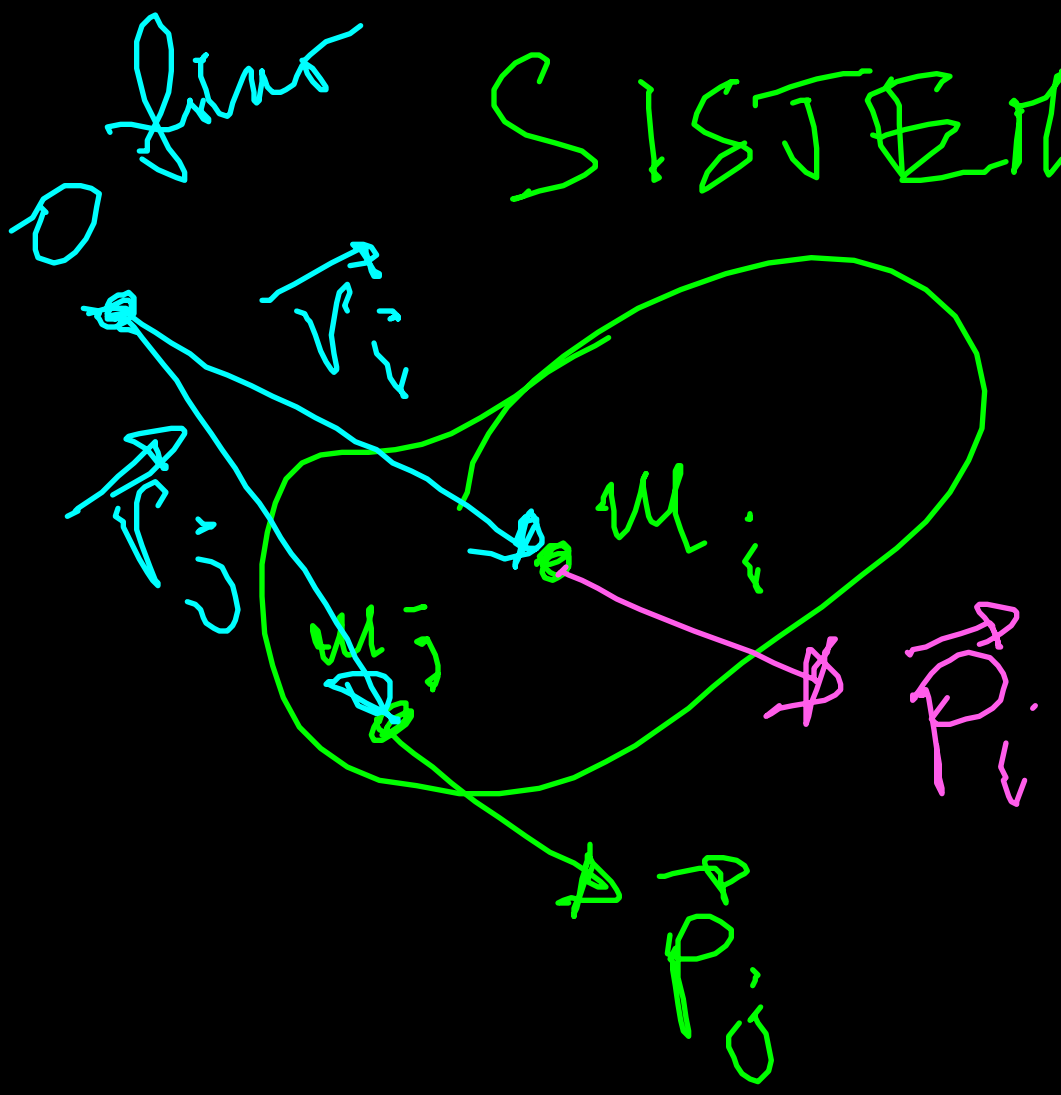
$$l_{O'} = (\vec{r}_P - \vec{r}_{O'}) \times \vec{p}$$

$$\frac{d\vec{l}_{O'}}{dt} = \frac{d}{dt} \left(\vec{r}_P \times \vec{p} + \vec{r}_{O'} \times \vec{p} \right)$$

$$\frac{d\vec{l}_{O'}}{dt} = \vec{r}_P \times \left(\sum \vec{F} \right) - \frac{d\vec{r}_{O'}}{dt} \times \vec{p}$$

POLO O' MOBILE

SISTEMA DI PUNTI MATERIALI



$$l_i = \vec{r}_i \times \vec{p}_i$$

$$\vec{L}_0 = \sum_{i=1}^n \vec{l}_i$$

MOM.
ANGOLARE
TOTALE
(rispetto ad O)

$$\frac{d\vec{L}_0}{dt} = \frac{d}{dt} \left(\sum \vec{l}_i \right)$$

nel punto "i"
equazione più forte

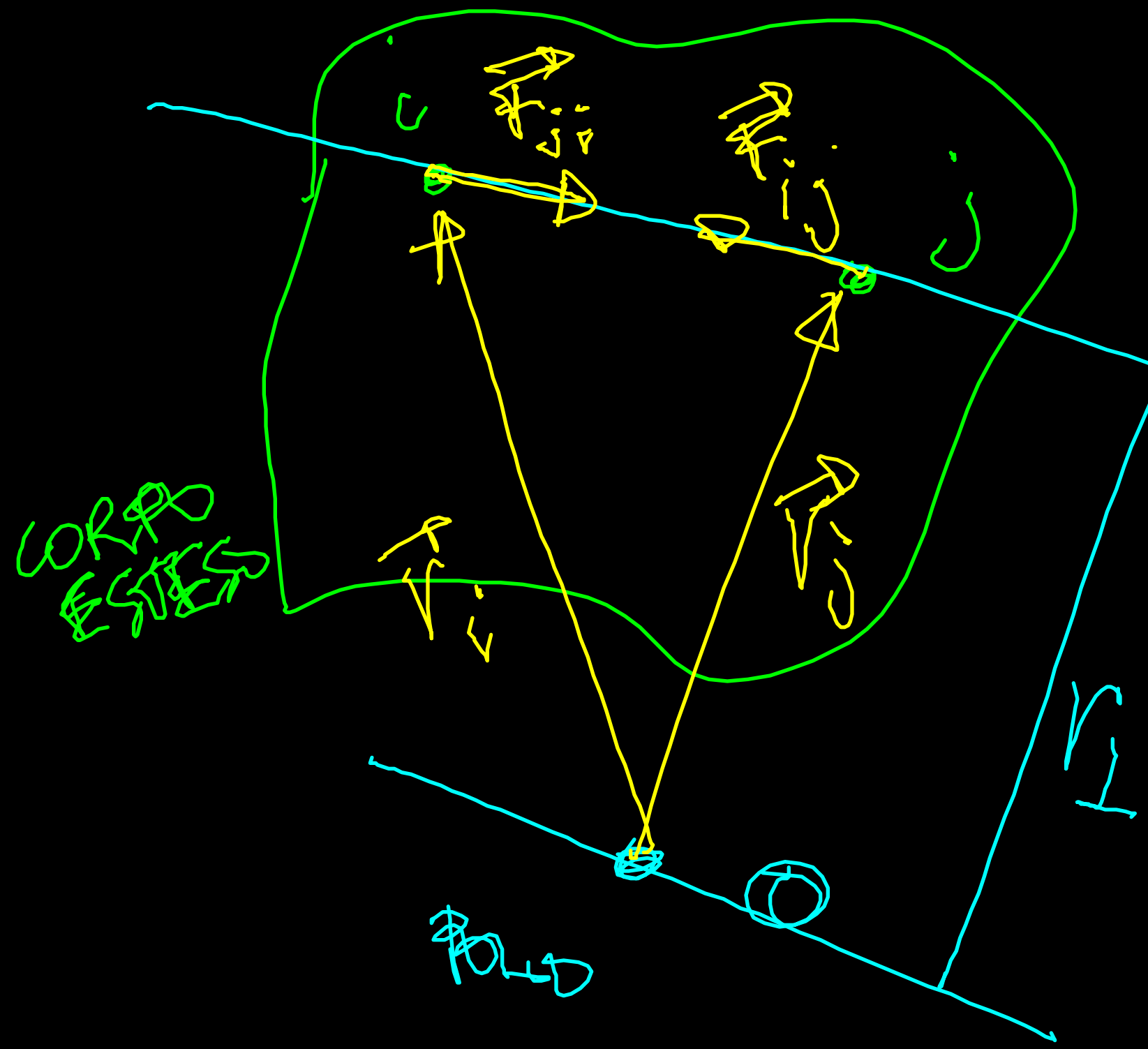
$$\frac{d\vec{l}_i}{dt} = \vec{r}_i \times \left(\sum \vec{F}_i \right)$$

$$= \vec{r}_i \times \vec{F}_i = \vec{L}_i = \vec{L}_{i,ext} + \vec{L}_{i,int}$$

momento delle forze
risultante applicato ad "i"

$$= \sum \frac{d\vec{L}_i}{dt} = \sum \vec{\tau}_i = \frac{d\vec{L}_0}{dt}$$

FORZE INTERNE



$$F_{ij} = -F_{ji}$$

$$F_{i0} = -F_{0i}$$

$$F_{ij} = -F_{ji}$$

$$\begin{aligned} \vec{r}_{ij} \cdot \vec{F}_{ij} &= r_{ij} F_{ij} \\ \vec{r}_{ji} \cdot \vec{F}_{ji} &= r_{ji} F_{ji} \end{aligned}$$

Veri opposti $\Rightarrow \vec{r}_{ij} = -\vec{r}_{ji}$

Per coppia di punti i e j $\vec{r}_{ij} + \vec{r}_{ji} = 0$

$$\Rightarrow \sum \vec{r}_{i,INT} = 0$$

$$\frac{dL_0}{dt} = \sum \left(\cancel{\vec{r}_{i,ext}} + \vec{r}_{i,ext} \right) = \sum \vec{r}_{i,ext}$$

